# Research Article <br> Kinetic Analysis of Omnidirectional Mobile Robot with Symmetry Roller's Arrangement 

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#### Abstract

In recent years, the efficient mobility of omnidirectional mobile robots is desired in the logistics industry. Hence, various types of rollers are installed in mobile robots. These include omnidirectional rollers, which are characterized with excellent omnidirectional mobility and are considered easy to control. Herein, considering that the side surface of the mechanism is circular, the kinematics of the mechanism assuming an arbitrary arrangement on this circle is derived. Previous research has evaluated the roller arrangement from the perspective of the speed efficiency of the drive roller. Meanwhile, this research focuses on a mobile robot equipped with three omni-rollers. Furthermore, focusing only on the translational speed of the robot, its motion efficiency is evaluated. The distribution of the robot's translational speed is studied, the area of the region is used as the evaluation function, and its behaviors are analyzed.


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## 1. Introduction

Recently, industries, such as logistics, have been looking forward to the practicality of using omnidirectional moving mechanisms, and several mobile robots have been developed with this aim. Holonomic mobile robots have attracted attention owing to their ease of control. This is because omnidirectional movement has three degrees of freedom, expressed as the sum of a twodimensional translational component and a onedimensional rotational component, which can be controlled independently. Thus, many movement mechanisms (with three omni-rollers) have been developed, and research on its kinematics and control has been conducted [1].
In this research, the RoboCup meddle-size-league soccer robot is characterized with an omnidirectional moving mechanism. The RV-infinity [2], Musashi150 [3], and NuBot [4] use the most basic triangular arrangement (equilateral triangle). Although no theoretical studies have been conducted on roller arrangements, studies on sphere transport [5], [6] and moving mechanisms [7], [8],
[9] have been reported in recent times, focusing on the driving rollers.

In moving mechanisms that use spheres as wheels (or mechanisms that use spheres as transport objects), the optimum angle of the sphere rotation axis with respect to movement direction has been determined, assuming an omnidirectional movement [7]. Furthermore, the optimum location for the two rollers driving the sphere is demonstrated to be at the equator [8].

Meanwhile, in the mobile mechanism that uses rollers as wheels, kinematics is represented using a linear transformation matrix, which maps the input (roller velocity) to the output (robot velocity) (translation and rotation components) [9]. Additionally, mechanisms with a line-symmetrical roller arrangement, as shown in Fig. 1 (b)(a)(c), are analyzed. In [10], translational and rotational motions are analyzed by adopting the "volume" of the image through first-order transformations and "area of positive projection" as criteria for evaluating the speed efficiency in movement.

In [11], an evaluation function was derived for a threeroller configuration with a line-symmetrical structure and was used to analyze only translational motion. However, the analysis for mechanisms with an origin-symmetrical
roller arrangement, as shown in Fig.1(d)(a)(e), remains unsolved.

In this study, the efficiency of the mobile robot (equipped with three omni-rollers), as shown in Fig. 1 (d)(a)(e), is investigated for only the translational movement. For this purpose, an evaluation function is derived for the general arrangement. The distribution of the translational velocity of the robot is also investigated, and the area of the region is used as an evaluation function to analyze the robot's behaviors.

The rest of this study is as follows. Section 2 discusses the kinematics of mobile robots corresponding to transformation mapping. Section 3 derives a more general sectional area function. Section 4 conducts the simulation. Finally, Section 5 presents the summary and future tasks.


Fig. 1 Mobile robots with three omni-roller arrangement (b)(a)(c), line-symmetry roller arrangement. (d)(e), and point-symmetry roller arrangement.

## 2. Transformation mapping and kinematics

As shown in Fig. 2(a), the mobile robot with a common radius $L$ adapted the $i$-th rollers $(i=1,2,3)$ at contact point $\boldsymbol{P}_{\boldsymbol{i}}$, which is positioned as angle $\theta_{i}$ on a global $X-Y$ coordinate system (origin $\boldsymbol{O}$ ).

Here, the roller peripheral speed $\left[v_{1}, v_{2}, v_{3}\right]^{T}$ is given, while the robot translation speed $\boldsymbol{V}=\left[V_{x}, V_{y}\right]^{T}$ and rational speed $L \dot{\phi}(\dot{\phi}$ : robot angular velocity) can be calculated. Thus, $\left[V_{x}, V_{y}, L \dot{\phi}\right]^{T}$ is determined.

As shown in Fig. 2(b), correspondences of [ $v_{1}, v_{2}, v_{3}$ ] and $\left[V_{x}, V_{y}, \dot{\phi} L\right]$ are represented using linear transformation mapping as $f_{A}:\left[V_{x}, V_{y}, \dot{\phi} L\right]^{T} \rightarrow$ $\left[v_{1}, v_{2}, v_{3}\right]^{T} . f_{A^{-1}}(W)=$ Image $f_{A^{-1}} \in \mathbf{R}^{3}$ is a parallelepiped domain from cubic domain $W$.

Furthermore, the sectional domain of $f_{A^{-1}}(W)$ for horizontal plane $\left\{V_{x} V_{y}\right.$ - plane $\}$ is as follow.


Fig. 2 Correspondence omni-roller speed $\left[v_{1}, v_{2}, v_{3}\right]^{T}$ and robot speed elements $\left[V_{x}, V_{y}, \dot{\phi} L\right]^{T}$ : (a). mobile robot and (b). transformation mapping.

$$
\begin{align*}
& f_{A^{-1}}(W) \cap\left\{V_{x} V_{y} \text { - plane }\right\} \\
& \quad=\left\{\left(V_{x}, V_{y}, 0\right)| | v_{1}\left|,\left|v_{2}\right|,\left|v_{3}\right| \leq 1\right\}\right. \tag{1}
\end{align*}
$$

Where

$$
\begin{align*}
f_{A^{-1}}(W) & =\left\{\left(V_{x}, V_{y}, L \dot{\phi}\right)| | v_{1}\left|,\left|v_{2}\right|,\left|v_{3}\right| \leq 1\right\}\right.  \tag{2}\\
& =\left\{\left(v_{1}, v_{2}, v_{3}\right)| | v_{1}\left|,\left|v_{2}\right|,\left|v_{3}\right| \leq 1\right\}\right. \tag{3}
\end{align*}
$$

Inverse kinematics $\left(f_{A}:\left[V_{x}, V_{y}, \dot{\phi} L\right]^{\boldsymbol{T}} \rightarrow\right.$ $\left[v_{1}, v_{2}, v_{3}\right]^{T}$ ) is represented as the following Eq. (4).

$$
\left[\begin{array}{l}
v_{1}  \tag{4}\\
v_{2} \\
v_{3}
\end{array}\right]=\left[\begin{array}{lll}
-\sin \theta_{1} & \cos \theta_{1} & 1 \\
-\sin \theta_{2} & \cos \theta_{2} & 1 \\
-\sin \theta_{3} & \cos \theta_{3} & 1
\end{array}\right]\left[\begin{array}{c}
V_{x} \\
V_{y} \\
L \dot{\phi}
\end{array}\right]
$$

Additionally, forward kinematics ( $f_{A^{-1}}$ : $\left.\left[v_{1}, v_{2}, v_{3}\right]^{T} \rightarrow\left[V_{x}, V_{y}, \dot{\phi} L\right]^{T}\right)$ is represented using Eq. (5).

$$
\left[\begin{array}{c}
V_{x}  \tag{5}\\
V_{y} \\
L \dot{\phi}
\end{array}\right]=\frac{1}{\operatorname{det} \boldsymbol{A}}\left[\begin{array}{lll}
-\sin \theta_{1} & \cos \theta_{1} & 1 \\
-\sin \theta_{2} & \cos \theta_{2} & 1 \\
-\sin \theta_{3} & \cos \theta_{3} & 1
\end{array}\right]^{-1}\left[\begin{array}{l}
v_{1} \\
v_{2} \\
v_{3}
\end{array}\right]
$$

## 3. Analysis for sectional shape and area

As a special case, a sectional area is analyzed in the case of isosceles triangle-like-three-roller arrangement in [11]. Meanwhile, in this study, a more general case is analyzed regarding sectional shapes.

### 3.1. Setup for domain and division

Cubic domain $W$ is composed following 8 -apexes $\boldsymbol{P}=(1,1,1), \boldsymbol{Q}=(-1,1,1), \boldsymbol{R}=(-1,-1,1), \boldsymbol{S}=$ $(1,-1,1), \boldsymbol{P}=(1,1,-1), \dot{Q}=(-1,1,-1), \dot{\boldsymbol{R}}=(-1$, $-1,-1), \boldsymbol{S}=(1,-1,-1)$.

Focusing on $L \dot{\phi}$-component of Eq. (5), we define $\theta_{3}=$ $0^{\circ}$.

$$
F\left(v_{1}, v_{2}, v_{3}\right)=\left[\begin{array}{l}
v_{1}  \tag{6}\\
v_{2} \\
v_{3}
\end{array}\right] \cdot\left[\begin{array}{l}
n_{1} \\
n_{2} \\
n_{3}
\end{array}\right]
$$

Where

$$
\begin{gather*}
n_{1}=-\sin \theta_{2}  \tag{7}\\
n_{2}=\sin \theta_{1}  \tag{8}\\
n_{3}=-\sin \left(\theta_{1}-\theta_{2}\right) \tag{9}
\end{gather*}
$$

In the case of translational motion, $\dot{\phi}=0$ is equivalent to $F\left(v_{1}, v_{2}, v_{3}\right)=0$, which is a linear equation with respect to $v_{1}, v_{2}$, and $v_{3}$. Therefore, geometrically, it can be represented as a plane with an origin and normal vector $\boldsymbol{n}=\left[n_{1}, n_{2}, n_{3}\right]^{T}$. Thus, the arrangement position of roller is completely dependent on $\boldsymbol{n}$.

As shown in Fig. 3, $D=\left\{\left(v_{1}, v_{2}\right)| | v_{1}\left|,\left|v_{2}\right| \leq 1\right\}\right.$ is a sectional domain of $W$ by $\nu_{3}=0$ and can be decomposed to the following four parts: $D_{++}, D_{-+}, D_{--}$, and $D_{+-}$.

$$
\begin{align*}
& D_{++}=\left\{\left(v_{1}, v_{2}\right) \mid 0<v_{1} \leq 1,0<v_{2} \leq 1\right\}  \tag{10}\\
& D_{-+}=\left\{\left(v_{1}, v_{2}\right) \mid-1<v_{1} \leq 0,0<v_{2} \leq 1\right\}  \tag{11}\\
& D_{--}=\left\{\left(v_{1}, v_{2}\right) \mid-1<v_{1} \leq 0,-1<v_{2} \leq 0\right\}  \tag{12}\\
& D_{+-}=\left\{\left(v_{1}, v_{2}\right) \mid 0<v_{1} \leq 1,-1<v_{2} \leq 0\right\} \tag{13}
\end{align*}
$$

In this study, we focus on domain $D_{++}$(grey highlight in Fig. 3). It can be decomposed into two parts by a sign of $n_{3}$. From $0^{\circ}<\theta_{1}, \theta_{2}<360^{\circ}$, and $\theta_{1} \neq \theta_{2},\left(\theta_{1}, \theta_{2}\right)$ is satisfied with following inequality.

$$
\begin{equation*}
-360^{\circ}<\theta_{1}-\theta_{2}<360^{\circ}, \theta_{1} \neq \theta_{2} \tag{14}
\end{equation*}
$$

In the case of $n_{3}>0$, it is equivalent to $180^{\circ}<\theta_{1}-$ $\theta_{2}<360^{\circ}$ and $-180^{\circ}<\theta_{1}-\theta_{2}<0^{\circ}\left(-360^{\circ}+\theta_{1}<\right.$ $\left.\theta_{2}<-180^{\circ}+\theta_{1}\right)$. Thus, the following inequality is obtained.

$$
\begin{equation*}
\theta_{1}<\theta_{2}<180^{\circ}+\theta_{1} \tag{15}
\end{equation*}
$$

In the case of $n_{3}<0$, it is equivalent to $0^{\circ}<\theta_{1}-$ $\theta_{2}<360^{\circ}$ and $-360^{\circ}<\theta_{1}-\theta_{2}<-180^{\circ}\left(-180^{\circ}+\right.$ $\theta_{1}<\theta_{2}<\theta_{1}$ ). Thus, the following inequality is given.

$$
\begin{equation*}
180^{\circ}+\theta_{1}<\theta_{2}<360^{\circ}+\theta_{1} \tag{16}
\end{equation*}
$$

Using Eq. (15) and Eq. (16), $D_{++}$can be decomposed to domains $D_{++}^{-}$and $D_{++}^{+}$.

$$
\begin{align*}
& D_{++}^{-}  \tag{17}\\
& =\left\{\left(\theta_{1}, \theta_{2}\right) \mid \theta_{1}<\theta_{2} \leq 360^{\circ}, 0^{\circ}<\theta_{1} \leq 180^{\circ}\right\} \\
& D_{++}^{+} \\
& =\left\{\left(\theta_{1}, \theta_{2}\right) \mid 180^{\circ}<\theta_{2} \leq \theta_{1}, 0^{\circ}<\theta_{1} \leq 180^{\circ}\right\} \tag{18}
\end{align*}
$$



Fig. 3 Three-roller arrangement poteen corresponding to domain $D_{++}^{-}$and $D_{++}^{+}$.


Fig. 4 Classification of the sectional shape of a cube in the case of domain $D_{++}^{-}$: (a). hexagon and (b). rhombus shapes.


Fig. 5 Classification of sectional in $D_{++}$(a). The case of hexagon domain $D_{++}^{-}$(b), hexagon domain $D_{++}^{+}$(c), and rhombus domain $D_{++}$

### 3.2. Analysis of sectional shapes

The sectional shape condition in $D_{++}^{-}$is considered. From Eq. (17), $\left(\theta_{1}, \theta_{2}\right)$ is satisfied because $0^{\circ}<\theta_{1} \leq$ $180^{\circ}$ and $180^{\circ}<\theta_{2} \leq 360^{\circ}$.

In cases where $n_{3} \neq 0$ and $\theta_{2}-\theta_{1}<180^{\circ}$, $F(1,1,1)$ and $F(1,1,-1)$ are calculated as follows:

$$
\begin{align*}
& F(1,1,1)=\sin \theta_{1}-\sin \theta_{2}-\sin \left(\theta_{1}-\theta_{2}\right)>0  \tag{19}\\
& F(1,1,-1)=\sin \theta_{1}-\sin \theta_{2}+\sin \left(\theta_{1}-\theta_{2}\right)  \tag{20}\\
& =\sin \theta_{1}+\sin \left(-\theta_{2}\right)+\sin \left(\theta_{1}+\left(-\theta_{2}\right)\right)  \tag{21}\\
& =\sin \theta_{1}+\sin \left(-\theta_{2}\right)+\sin \theta_{1} \cos \left(-\theta_{2}\right) \\
& +\cos \theta_{1} \sin \left(-\theta_{2}\right) \tag{22}
\end{align*}
$$

From $1+\cos \left(-\theta_{2}\right)>0$ and $1+\cos \theta_{1}>0$,

$$
\begin{align*}
=\sin \theta_{1}(1 & \left.+\cos \left(-\theta_{2}\right)\right) \\
& +\sin \left(-\theta_{2}\right)\left(1+\cos \theta_{1}\right)>0 \tag{23}
\end{align*}
$$

Thus, $\boldsymbol{P}(1,1,1)$ and $\boldsymbol{P}(1,1,-1)$ are up word on of plane's normal vector direction. Thus, the sectional shape of the domain is hexagon (Fig. 4(a)).

In the case where $n_{3}=0, \boldsymbol{n}$ is parallel to $\left\{V_{x} V_{y}-\right.$ plane $\}$. Thus, the sectional shape of the domain is rhombus (Fig. 4(b)).
$\pi \cap W$ is satisfied as the following property.

## Property1

The sectional shape has a line symmetry with respect to the origin and can be divided into the following two classes:
$\left|\theta_{1}-\theta_{2}\right| \neq 180^{\circ} \Rightarrow$ Hexagon
$\left|\theta_{1}-\theta_{2}\right|=180^{\circ} \Rightarrow$ Rhombus

### 3.3. Derivation of sectional area function

$\pi \cap W=$ Hexagon $\quad \boldsymbol{T}_{\boldsymbol{R S}} \boldsymbol{T}_{\boldsymbol{S} \boldsymbol{S}} \boldsymbol{T}_{\hat{P} \dot{S}} \boldsymbol{T}_{\hat{P} \dot{Q}} \boldsymbol{T}_{Q \dot{Q}} \boldsymbol{T}_{Q \boldsymbol{R}} \quad$ is decomposed to the following six parts: $\triangle \boldsymbol{O} \boldsymbol{T}_{\boldsymbol{R} S} \boldsymbol{T}_{Q R}, \triangle$ $O T_{Q R} T_{Q \dot{Q}}, \triangle O T_{Q \dot{Q}} T_{R S}, \triangle O T_{\hat{P} \dot{S}} T_{\tilde{P Q}}, \triangle O T_{\tilde{P} \dot{S}} T_{S S}$, and $\triangle \boldsymbol{O} \boldsymbol{T}_{Q \dot{Q}} \boldsymbol{T}_{\boldsymbol{P} \dot{Q} \dot{Q}}$. Furthermore, the transformation mapping of $f_{A^{-1}}$ shows that $f_{A^{-1}}(\boldsymbol{O})=\boldsymbol{O}$ and $f_{A^{-1}}(\pi \cap W)$ is represented as sum of $\triangle$ $\boldsymbol{O} f_{A^{-1}}\left(\boldsymbol{T}_{R S}\right) f_{A^{-1}}\left(\boldsymbol{T}_{Q R}\right), \Delta \boldsymbol{O} f_{A^{-1}}\left(\boldsymbol{T}_{Q R}\right) f_{A^{-1}}\left(\boldsymbol{T}_{Q \dot{Q}}\right), \Delta$ $\boldsymbol{O} f_{A^{-1}}\left(\boldsymbol{T}_{Q \dot{Q}}\right) f_{A^{-1}}\left(\boldsymbol{T}_{\boldsymbol{R} S}\right), \Delta \boldsymbol{O} f_{A^{-1}}\left(\boldsymbol{T}_{\dot{P} \dot{S} \dot{S}}\right) f_{A^{-1}}\left(\boldsymbol{T}_{\dot{P} \dot{Q}}\right), \Delta$ $\boldsymbol{O} f_{A^{-1}}\left(\boldsymbol{T}_{\dot{P} \dot{S}}\right) f_{A^{-1}}\left(\boldsymbol{T}_{S \dot{S}}\right)$, and $\triangle \boldsymbol{O} f_{A^{-1}}\left(\boldsymbol{T}_{Q \dot{Q}}\right) f_{A^{-1}}\left(\boldsymbol{T}_{\dot{P} \dot{Q}}\right)$.

Area of $f_{A^{-1}}(\pi \cap W)$ is defined as $D_{S e c}\left(\theta_{1}, \theta_{2}\right)$ $\left(\left(\theta_{1}, \theta_{2}\right) \in D_{++}\right)$. It is decomposed into the following $D_{\text {Sec }}^{+}\left(\theta_{1}, \theta_{2}\right) \quad\left(\left(\theta_{1}, \theta_{2}\right) \in D_{++}^{+}\right)$and $D_{S e c}^{-}\left(\theta_{1}, \theta_{2}\right)$ $\left(\left(\theta_{1}, \theta_{2}\right) \in D_{++}^{-}\right)$, which are function depended on roller contact position ( $\theta_{1}, \theta_{2}, 0^{\circ}$ ).

The opposite sides are parallel and have the same length. Thus, hexagon $f_{A^{-1}}(\pi \cap W)$ has a symmetry with respect to origin $\boldsymbol{O}$. Thus, in the case of hexagon, $D_{\text {Sec }}^{+}\left(\theta_{1}, \theta_{2}\right)$ and $D_{\text {Sec }}^{-}\left(\theta_{1}, \theta_{2}\right)$ are derived as the sum of
three triangle areas (Fig. 5(a)(b)). Furthermore, in the case of rhombus, $D_{\text {Sec }}\left(\theta_{1}, \theta_{2}\right)$ is derived as the sum of the same eight triangle areas (Fig. 5(c)).

$$
\begin{align*}
& D_{S e c}^{+}\left(\theta_{1}, \theta_{2}\right)=\left\|f_{A^{-1}}\left(\boldsymbol{T}_{R S}\right) \times f_{A^{-1}}\left(\boldsymbol{T}_{Q R}\right)\right\| \\
&+\left\|f_{A^{-1}}\left(\boldsymbol{T}_{Q R}\right) \times f_{A^{-1}}\left(\boldsymbol{T}_{Q \dot{Q}}\right)\right\| \\
&+\left\|f_{A^{-1}}\left(\boldsymbol{T}_{\boldsymbol{S} \dot{S}}\right) \times-f_{A^{-1}}\left(-\boldsymbol{T}_{\boldsymbol{R S}}\right)\right\| \\
&\left(\left|\theta_{1}-\theta_{2}\right| \neq 180^{\circ},\left(\theta_{1}, \theta_{2}\right) \in D_{++}^{+}\right) \tag{24}
\end{align*}
$$

$$
D_{S e c}^{-}\left(\theta_{1}, \theta_{2}\right)=\left\|f_{A^{-1}}\left(\boldsymbol{T}_{\boldsymbol{P S}}\right) \times f_{A^{-1}}\left(\boldsymbol{T}_{\boldsymbol{Q P}}\right)\right\|
$$

$$
+\left\|f_{A^{-1}}\left(\boldsymbol{T}_{Q P}\right) b \times f_{A^{-1}}\left(\boldsymbol{T}_{Q \dot{Q}}\right)\right\|
$$

$$
+\left\|f_{A^{-1}}\left(\boldsymbol{T}_{\boldsymbol{S}}\right) \times-f_{A^{-1}}\left(-\boldsymbol{T}_{P S}\right)\right\|
$$

$$
\begin{equation*}
\left(\left|\theta_{1}-\theta_{2}\right| \neq 180^{\circ},\left(\theta_{1}, \theta_{2}\right) \in D_{++}^{-}\right) \tag{25}
\end{equation*}
$$

$$
D_{S e c}\left(\theta_{1}, \theta_{2}\right)=4\left\|f_{\boldsymbol{A}^{-1}}\left(\boldsymbol{T}_{\boldsymbol{R} S}\right) \times f_{\boldsymbol{A}^{-1}}(0,0,1)\right\|
$$

$$
\begin{equation*}
\left(\left|\theta_{1}-\theta_{2}\right|=180^{\circ},\left(\theta_{1}, \theta_{2}\right) \in D_{++}\right) \tag{26}
\end{equation*}
$$

Where $\boldsymbol{T}_{\boldsymbol{R} \boldsymbol{S}}\left(t_{R S},-1,1\right), \boldsymbol{T}_{Q \boldsymbol{R}}\left(-1, t_{Q R}, 1\right), \boldsymbol{T}_{\boldsymbol{S P}}\left(1, t_{S P}, 1\right)$, $\boldsymbol{T}_{\boldsymbol{P Q}}\left(t_{P Q}, 1,1\right)$, and $\boldsymbol{T}_{Q \dot{Q}}\left(-1,1, t_{Q \dot{Q}}\right)$.

$$
\begin{gather*}
t_{R S}=\frac{-\sin \theta_{1}-\sin \left(\theta_{1}-\theta_{2}\right)}{\sin \theta_{2}}  \tag{27}\\
t_{Q R}=\frac{-\sin \theta_{2}+\sin \left(\theta_{1}-\theta_{2}\right)}{\sin \theta_{1}}  \tag{28}\\
t_{S P}=\frac{\sin \theta_{2}+\sin \left(\theta_{1}-\theta_{2}\right)}{\sin \theta_{1}}  \tag{29}\\
t_{P Q}=\frac{\sin \theta_{1}-\sin \left(\theta_{1}-\theta_{2}\right)}{\sin \theta_{2}}  \tag{30}\\
t_{Q Q}=\frac{\sin \theta_{1}+\sin \theta_{2}}{\sin \left(\theta_{1}-\theta_{2}\right)} \tag{31}
\end{gather*}
$$

## 4. Simulation

This section presents the simulation findings, including the horizontal cross-sectional area $D_{S e c}$, as evaluation values.

Section 4.1 shows the case of a roller arrangement that has a line symmetry with respect to the $x$-axis.

Section 4.2 shows the case of a roller arrangement that has a point symmetry with respect to the origin.

Simulations were performed on three various roller arrangement patterns. Types (A)-(E) are arranged in the shape of a triangle, corresponding to Fig. 1(a)-(e), respectively.

### 4.1. Case of line-symmetry arrangement with respect to the $x$-axis

Fig. 6 (b)(a)(c) shows the shape in the case of a triangle that has a line symmetry with respect to the $x$-axis.
$D_{\text {Sec }}^{+}(\theta, 2 \pi-\theta)$ (substituting Eq. (24) to $\theta_{1}=\theta$ and $\theta_{2}=2 \pi-\theta$ for $\left.90^{\circ} \leq \theta<180^{\circ}\right)$ and $D_{\text {Sec }}^{-}(\theta, 2 \pi-\theta)$ (substituting Eq. (25) to $\theta_{1}=\theta$ and $\theta_{2}=2 \pi-\theta$ for $\left.0^{\circ} \leq \theta<90^{\circ}\right)$ are analyzed. They are set up as $\theta_{i}(i=$ $1,2,3)$ as follows:
Type(B): $\left(\theta_{1}, \theta_{2}, \theta_{3}\right)=\left(60^{\circ}, 300^{\circ}, 0^{\circ}\right)$
Type(A): $\left(\theta_{1}, \theta_{2}, \theta_{3}\right)=\left(90^{\circ}, 270^{\circ}, 0^{\circ}\right)$
Type(C): $\left(\theta_{1}, \theta_{2}, \theta_{3}\right)=\left(120^{\circ}, 240^{\circ}, 0^{\circ}\right)$ and $L=$ 1[m].

Fig. 6(d)(a)(e) show an outline of the velocity distributions for Type(A), Type(B), and Type(C). Their shapes are hexagon, square, and hexagon, respectively.

Fig. 7 shows the shape in the case of a triangle that has a point symmetry with respect to origin $\boldsymbol{O} \cdot D_{\text {Sec }}(\theta, 2 \pi-$ $\theta$ ) (substituting Eq. (26) to $\theta_{1}=\theta$ and $\theta_{2}=2 \pi-\theta$ for $\left.90^{\circ} \leq \theta<180^{\circ}\right)$, as a result, $D_{S e c}\left(90^{\circ}\right)=4.05[\mathrm{~m} / \mathrm{s}]^{2}$ and it takes minimal value $D_{S e c}\left(60^{\circ}\right)$ and $D_{S e c}\left(120^{\circ}\right)=$
$3.85[\mathrm{~m} / \mathrm{s}]^{2}$. Moreover, $D_{\text {Sec }}\left(0^{\circ}\right)=\infty$ and $D_{\text {Sec }}\left(180^{\circ}\right)=\infty$.

The result shows that Type(B) and Type(C) are the rollers arrangement with poorest efficiency for only translational motion.

### 4.2. Case of point-symmetry arrangement with respect

 to originFig. 6(d)(a)(e) shows the shape in the case of a triangle that has a point symmetry with respect to origin $\boldsymbol{O}$.
$D_{S e c}(\theta, 2 \pi-\theta)$ (substituting Eq. (26) to $\theta_{1}=\theta$ and $\theta_{2}=2 \pi+\theta$ for $0^{\circ} \leq \theta<180^{\circ}$ ) is analyzed and is set up as $\theta_{i}(i=1,2,3)$ as follows:

Type(D): $\left(\theta_{1}, \theta_{2}, \theta_{3}\right)=\left(60^{\circ}, 240^{\circ}, 0^{\circ}\right)$
Type(A): $\left(\theta_{1}, \theta_{2}, \theta_{3}\right)=\left(90^{\circ}, 270^{\circ}, 0^{\circ}\right)$
Type(E): $\left(\theta_{1}, \theta_{2}, \theta_{3}\right)=\left(120^{\circ}, 300^{\circ}, 0^{\circ}\right)$
Fig. 6 shows an outline of velocity distributions for Type(D), Type(A), and Type(E), represented as hexagon, square, and hexagon, respectively.
As shown in Fig. 8, it is symmetry function and takes minimal value $D_{S e c}\left(90^{\circ}\right)=4.05[\mathrm{~m} / \mathrm{s}]^{2}$. In addition, $D_{S e c}\left(0^{\circ}\right)$ and $D_{S e c}\left(180^{\circ}\right)=\infty$.
The result shows that Type(A) is the smallest and efficient when only translational motion is assumed.


Fig. 7 Behaviors of $D_{S e c}^{-}(\theta, 2 \pi-\theta)\left(0^{\circ} \leq \theta<180^{\circ}\right)$ and $D_{S e c}^{+}(\theta, 2 \pi-\theta)\left(90^{\circ} \leq \theta<180^{\circ}\right)$.


Fig. 8 Behaviors of $D_{\text {Sec }}(\theta, 2 \pi-\theta)\left(0^{\circ} \leq \theta<180^{\circ}\right)$


Fig. 6 Simulation result for the roller arrangement in the case of symmetry (a)(b)(c), line symmetry. (a)(d)(e), and point symmetry

## 5. Conclusion

In this study, a function was successfully derived to evaluate the efficiency of only the translational movement in a mobile mechanism equipped with three omni-rollers. Furthermore, the distribution of the translational velocity of the robot was analyzed through simulation, and the roller arrangement with the poorest efficiency was clarified.
The future works will focus on the efficiency of movement when the roller direction is changed arbitrarily in mobile robots.

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## Authors Introduction

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